

# Pilot-scale Investigation of Heat Flux and Radiation from an Oxy-coal Flame

## Part 1: Development of Instrument Models

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# Presentation Road Map

- Program objective, hierarchy and task objective
- Review of experimental quantities of interest
- New measurement devices with instrument models
  - Heat transfer surfaces
  - Wall thermocouples
  - Radiometers
- Example data set
- Summary & conclusions

# Project Objective

Implementation of exascale computing with V&V/UQ to more rapidly deploy a new technology for providing low cost, low emission electric power generation

V&V/UQ – Verification & Validation with Uncertainty Quantification

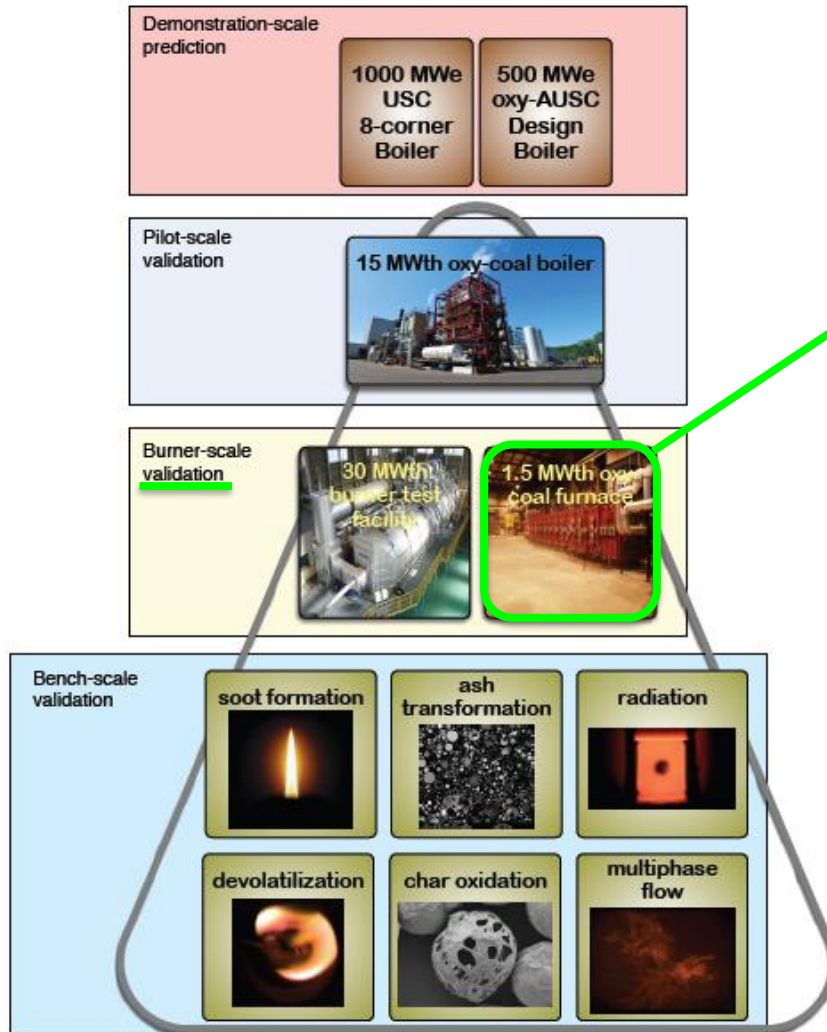


CARBON CAPTURE  
MULTIDISCIPLINARY  
SIMULATION CENTER

Ultimate goal to design a next-generation  
350 MWe oxy-coal boiler



# Program Hierarchy



1.5 MW pulverized coal furnace  
(L1500)

## Task Objectives

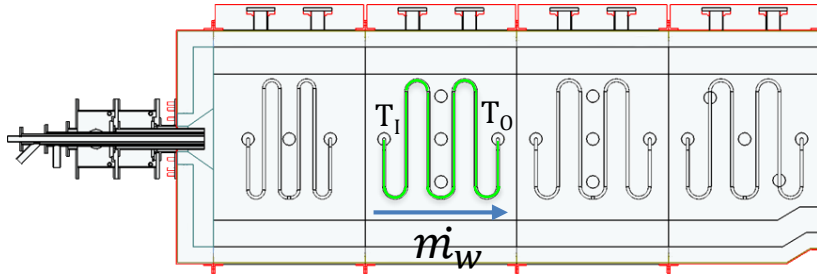
- Rework furnace measurement devices to accomplish the following:
  - Reduce the impact of measurement on the quantity of interest
  - Evaluate the relationship between the measured value and the quantity of interest
    - Simplify
    - Quantify through mathematical relationships (Instrument Model)
  - Assign value and uncertainty to the quantity of interest

## Quantities of Greatest Interest

- Heat removal through cooling surfaces
- Refractory temperatures at the flue gas interface
- Heat flux through the refractory walls
- Radiative intensity

# Measuring Heat Removal Through Cooling Surfaces

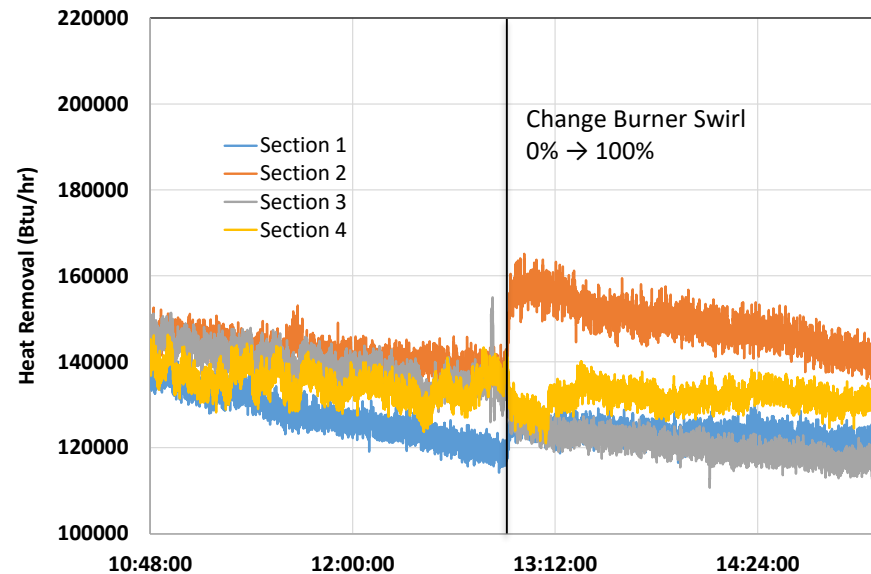
# Cooling Coils and Panels



- Cooling surfaces are necessary to provide steady state temperature profile
- Heat removal is determined by measuring the mass flow of water and the temperature of the water in and out

$$Q = \dot{m}_w \cdot c_p (T_O - T_I)$$

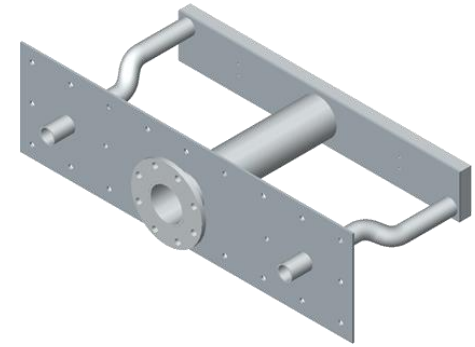
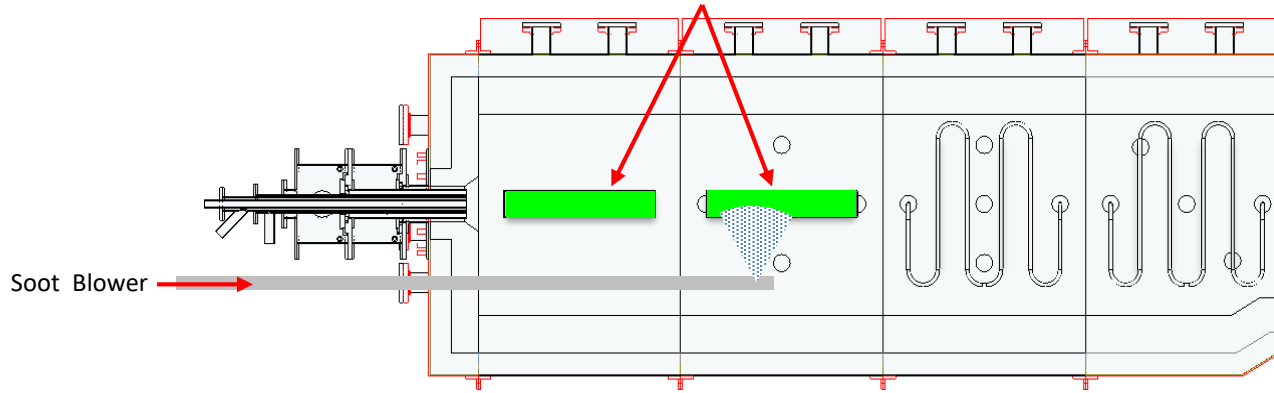
- Measurement is very sensitive to particle deposition





# Cooling Coils and Panels

Flat plate cooling panels



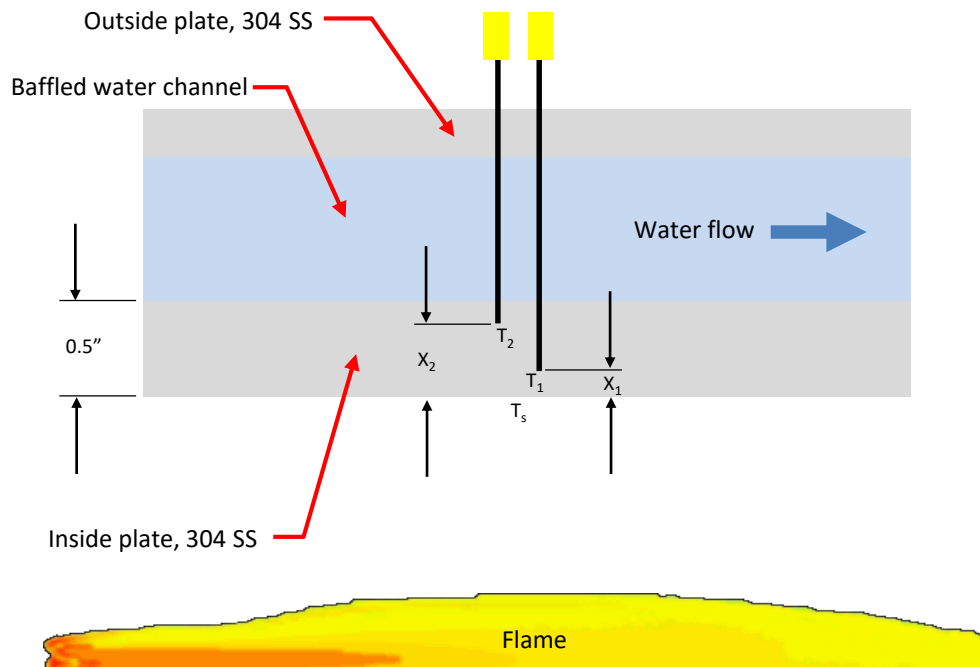
Multiple depth thermocouples placed in the hot-side plate for heat flux measurements

2 thermocouple sets / heat exchanger

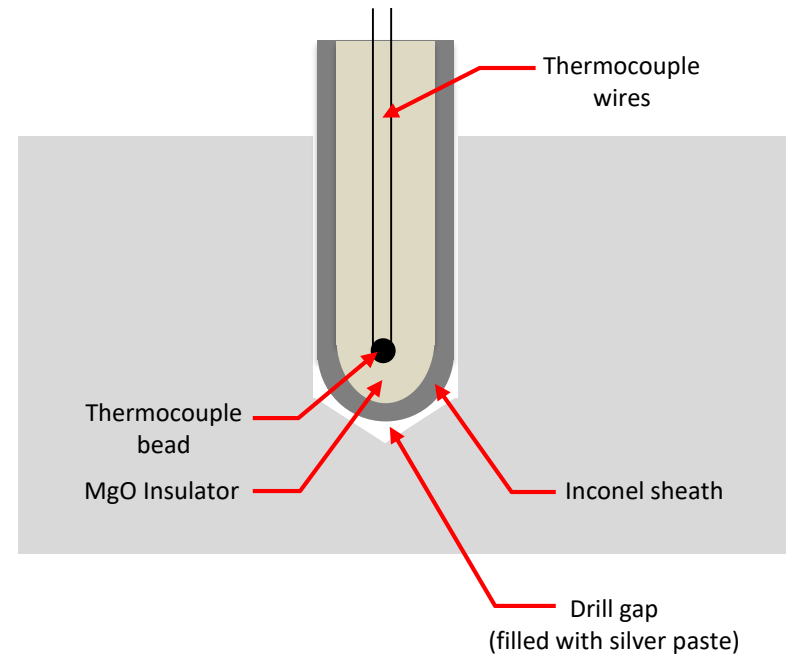
8 total heat flux measurements

# Cooling Coils and Panels

Cooling panel cross section



Thermocouple cross section



# Cooling Coils and Panels Instrument Model

Multi-depth thermocouple  
mathematical description:

$$q = k_{ref} \frac{(T_1 - T_2)}{(X_1 - X_2)} \quad T_s = T_1 + q \left( \frac{X_1}{K_{ref}} \right)$$

Assumption: The 1/16" thermocouple does not impact heat flux

Temperature profile within the thermocouple to bead

$$T_5 = T_1 - q \left[ \left( \frac{X_{Sil}}{K_{Sil}} \right) + \left( \frac{X_{inc}}{K_{inc}} \right) + \left( \frac{X_{MgO}}{K_{MgO}} \right) \right]$$

Assumption: Flux through plate = flux through thermocouple

Energy balance  
mathematical description:

$$Q = \dot{m}_w \cdot c_p (T_O - T_I)$$

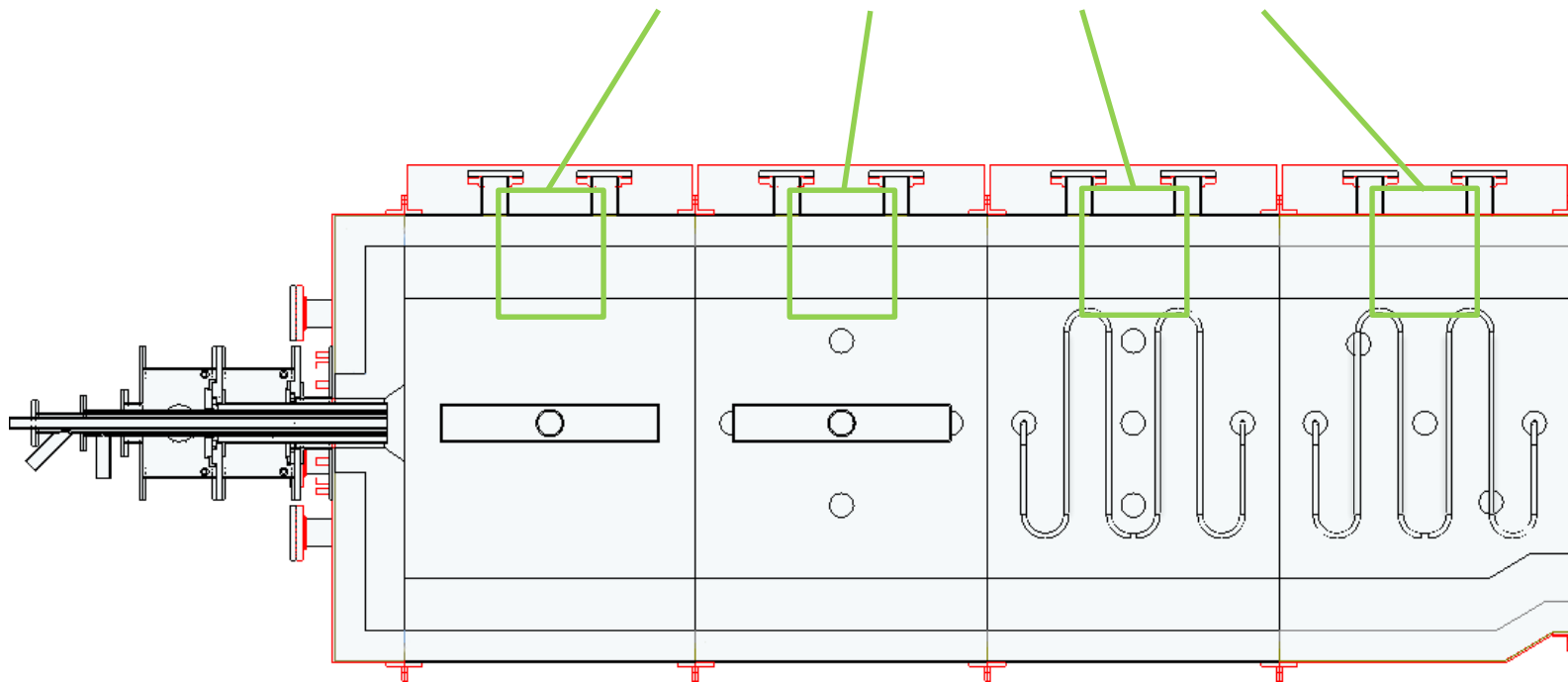
Quantifiable  
sources of error:

- Standard error in type-k thermocouple bead
- Variability in thermocouple set depth measurement
- Variability in material thermal properties
- Error in flow rate measurement

# Measuring Wall Temperatures and Wall Refractory Heat Flux

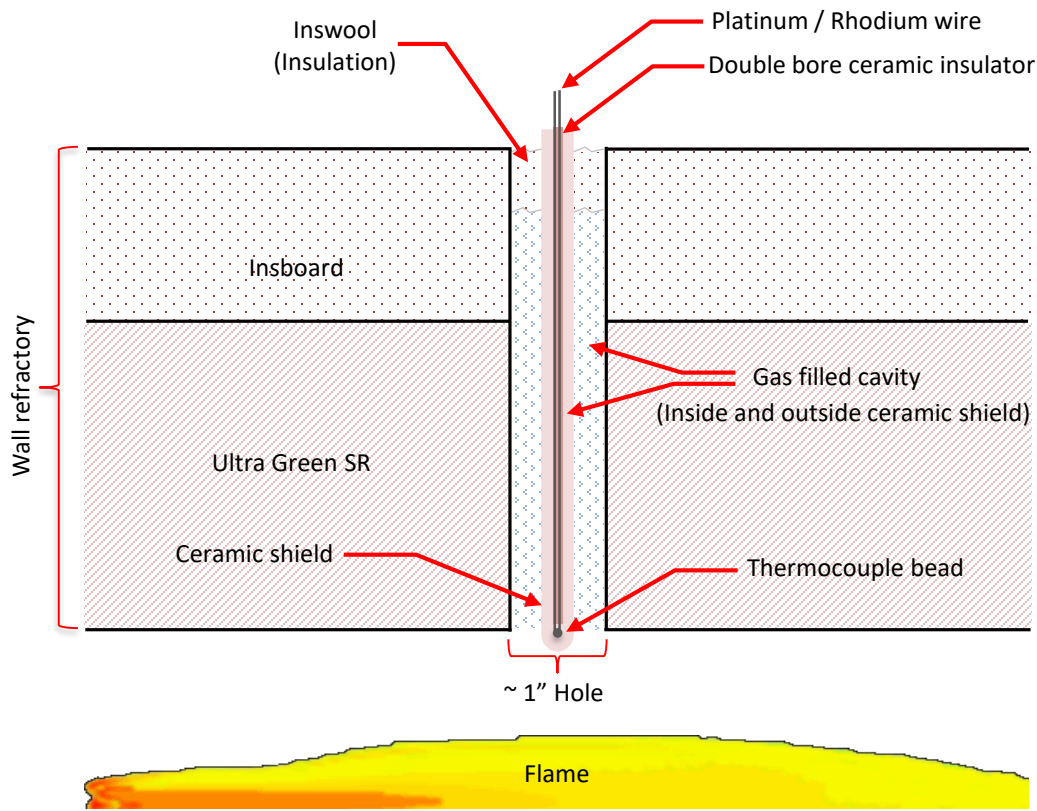
# Wall Thermocouples

Installed in the center of the top wall of each section



Permanently installed indicator of temperature profile  
(continuous data)

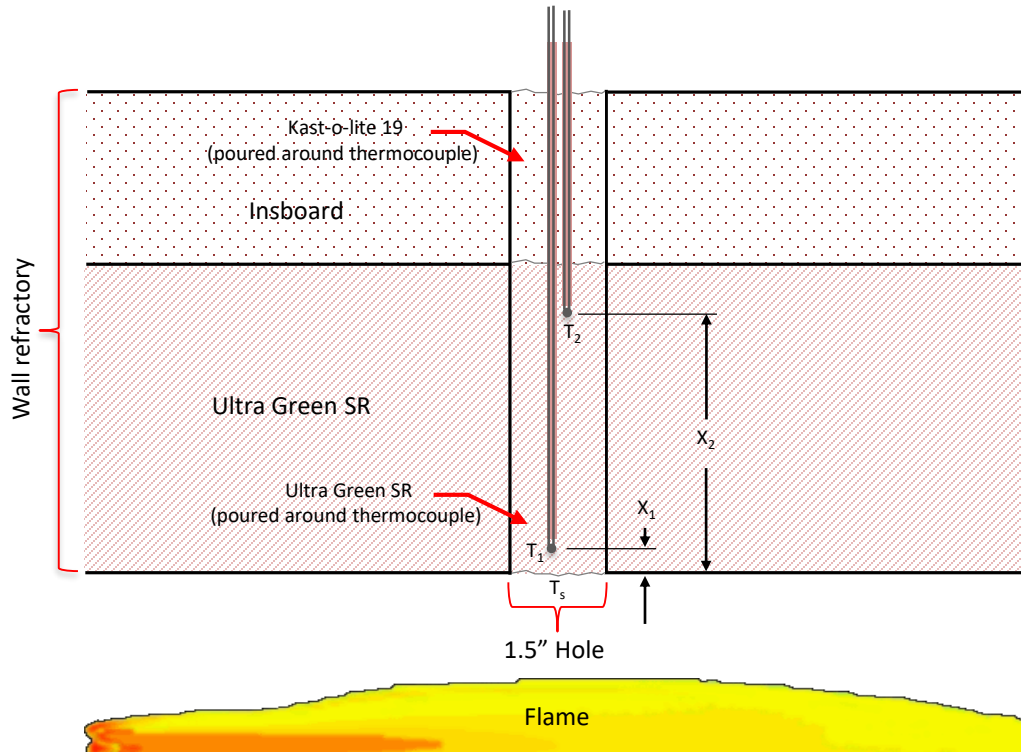
# Old Wall Thermocouple Device



## Measured temp is not of the wall

- Heat transfer characteristics of measurement device are dissimilar to surroundings
- Ceramic, wire and air gaps vs. refractory
- Placement of bead is uncertain
- Interpretation of the data requires a complicated model which includes the surrounding environment

# New Wall Thermocouple Device



## Advantages:

- Environment closely approximates the natural furnace wall
- Simple mathematical description of temperature profile
- Both surface temperature and heat flux can be acquired

## Disadvantages:

- Expensive
- Difficult to install

# New Wall Thermocouple Instrument Model

Mathematical  
Description:

$$q = k_{ref} \frac{(T_1 - T_2)}{(X_1 - X_2)} \quad T_s = T_1 + q \left( \frac{X_1}{K_{ref}} \right)$$

Assumption: The wire and double bore ceramic do not impact the temperature profile

Quantifiable  
sources of error:

- Standard error in type-B thermocouple bead
- Variability in thermocouple set depth measurement
- Variability in material thermal properties

Expected  
Behavior:

$$\Delta T = 748 \text{ to } 894 \pm 5 \text{ (}^\circ\text{C)}$$

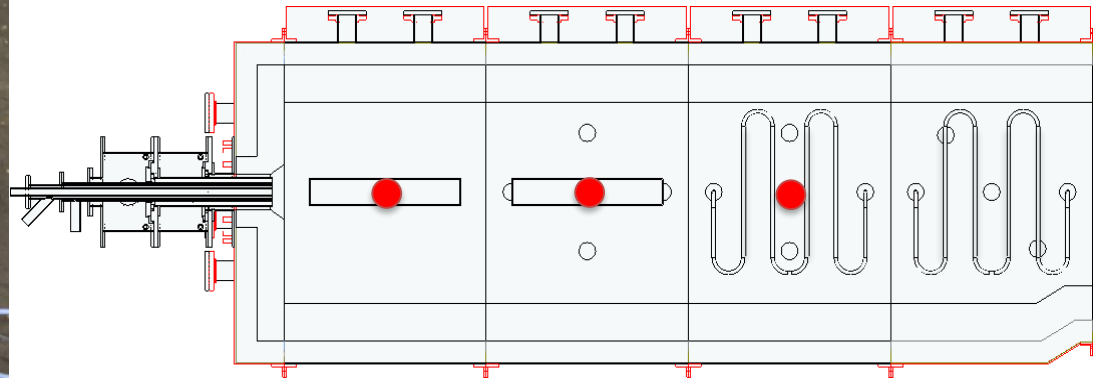
$$q = 1651 \text{ to } 1971 \pm 171 \text{ (W/m}^2\text{)}$$

Range is from section 1 through 10 device distributions



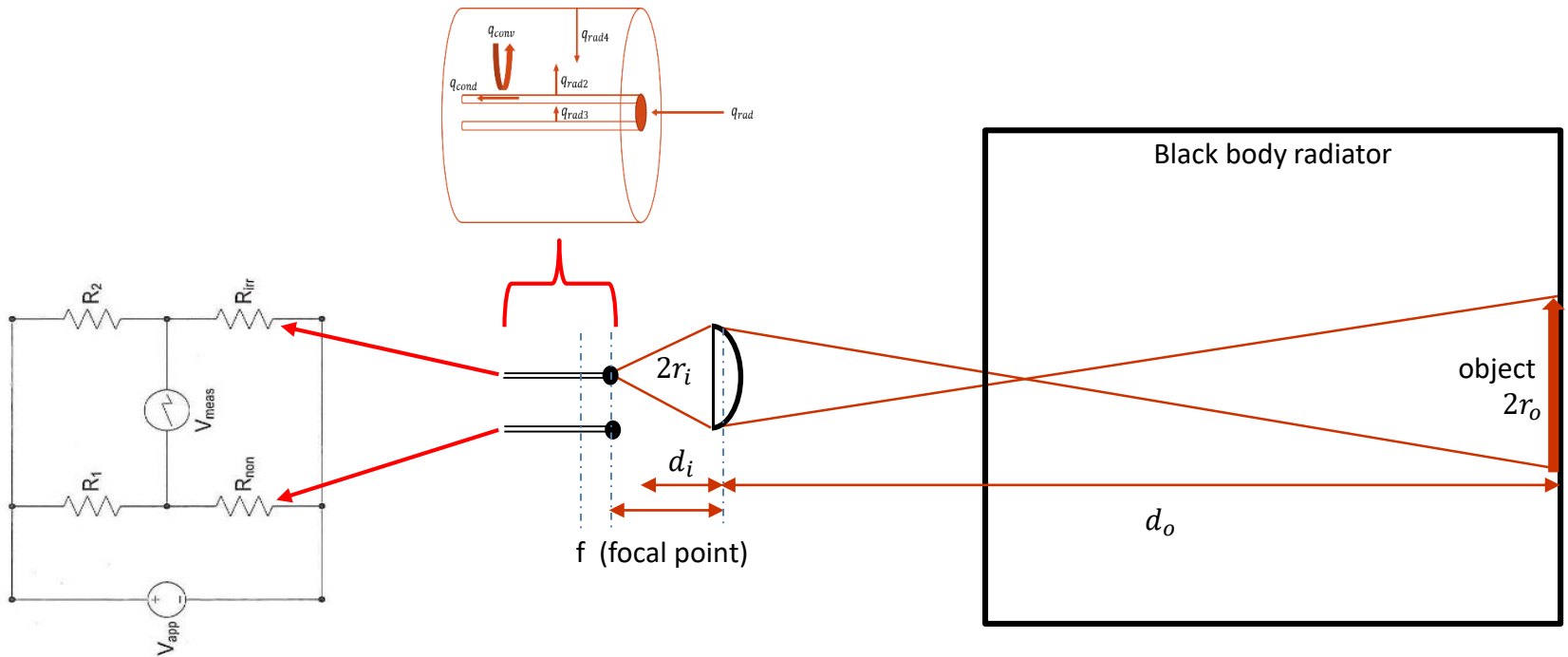
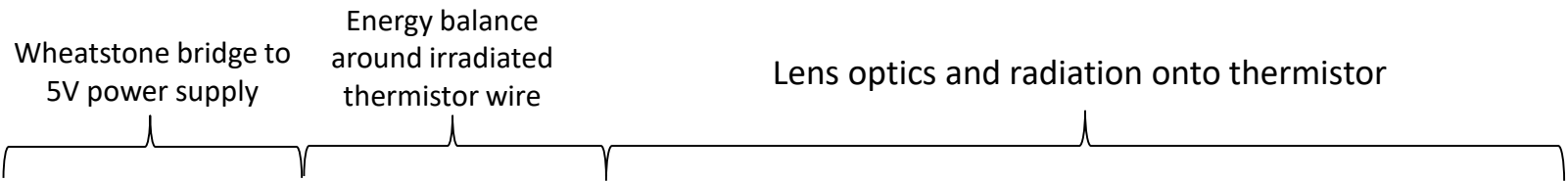
# Measuring Radiative Heat Flux

# Radiometer Configuration



- Installed on the center port in the first three sections of the furnace
- Open 4" cavity (optically dark) on the opposite side of the furnace
  - Minimize the wall effects and measure only flame properties

# Physical Processes of the Radiometer



# Radiometer Instrument Model

Mathematical  
Description:

$$d_i = \frac{1}{\frac{1}{d_o} + \frac{1}{f}} \quad r_i = \frac{d_i r_o}{d_o} \quad \left. \vphantom{\frac{1}{d_o} + \frac{1}{f}} \right\} \text{Lens optics}$$

$$I_i = I_o \left( \frac{r_{lens}}{r_i} \right) (1 - \rho) \quad \left. \vphantom{\left( \frac{r_{lens}}{r_i} \right)} \right\} \text{Thermistor irradiation}$$

$$q_{rad} = \pi r_i^2 I_i$$

$$q_{rad} + q_{rad3} + q_{rad4} = q_{cond} + q_{conv} + q_{rad2} \quad \left. \vphantom{q_{rad} + q_{rad3} + q_{rad4}} \right\} \text{Energy balance}$$

$$R_t = R_{ref} \exp \left( A + \frac{B}{T_t} + \frac{C}{T_t^2} + \frac{D}{T_t^3} \right)$$

$$V_{meas} = V_{app} \left( \frac{R_{non}}{R_{non} + R_1} - \frac{R_{irr}}{R_{irr} + R_2} \right) \quad \left. \vphantom{\frac{R_{non}}{R_{non} + R_1} - \frac{R_{irr}}{R_{irr} + R_2}} \right\} \text{Wheatstone bridge}$$

# **L1500 Heat Balance**

# L1500 Heat Balance ( High Temp Oxy-coal)

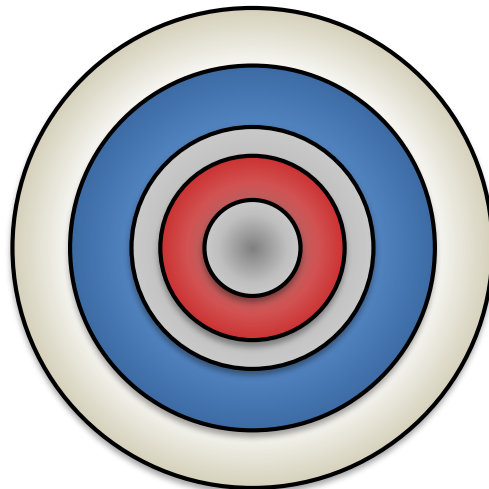
Targeted Conditions

Firing Rate	Btu/hr	3.0
Coal Rate	lb/hr	238
Primary Air/FGR	lb/hr	302
Primary O <sub>2</sub>	lb/hr	55
Inner Secondary Air/FGR	lb/hr	
Inner Secondary O <sub>2</sub>	lb/hr	478
Inner Secondary Temp	°F	100
Outer Secondary Air/FGR	lb/hr	
Outer Secondary O <sub>2</sub>	lb/hr	
Outer Secondary Temp	°F	

Skyline Coal Composition

C	70.60
H	5.05
N	1.42
S	0.53
O	10.39
Ash	8.83
Moisture	3.18
Volatile Matter	38.6
Fixed Carbon	49.4
HHV, Btu/lb	12606

\* all values in mass % unless otherwise specified

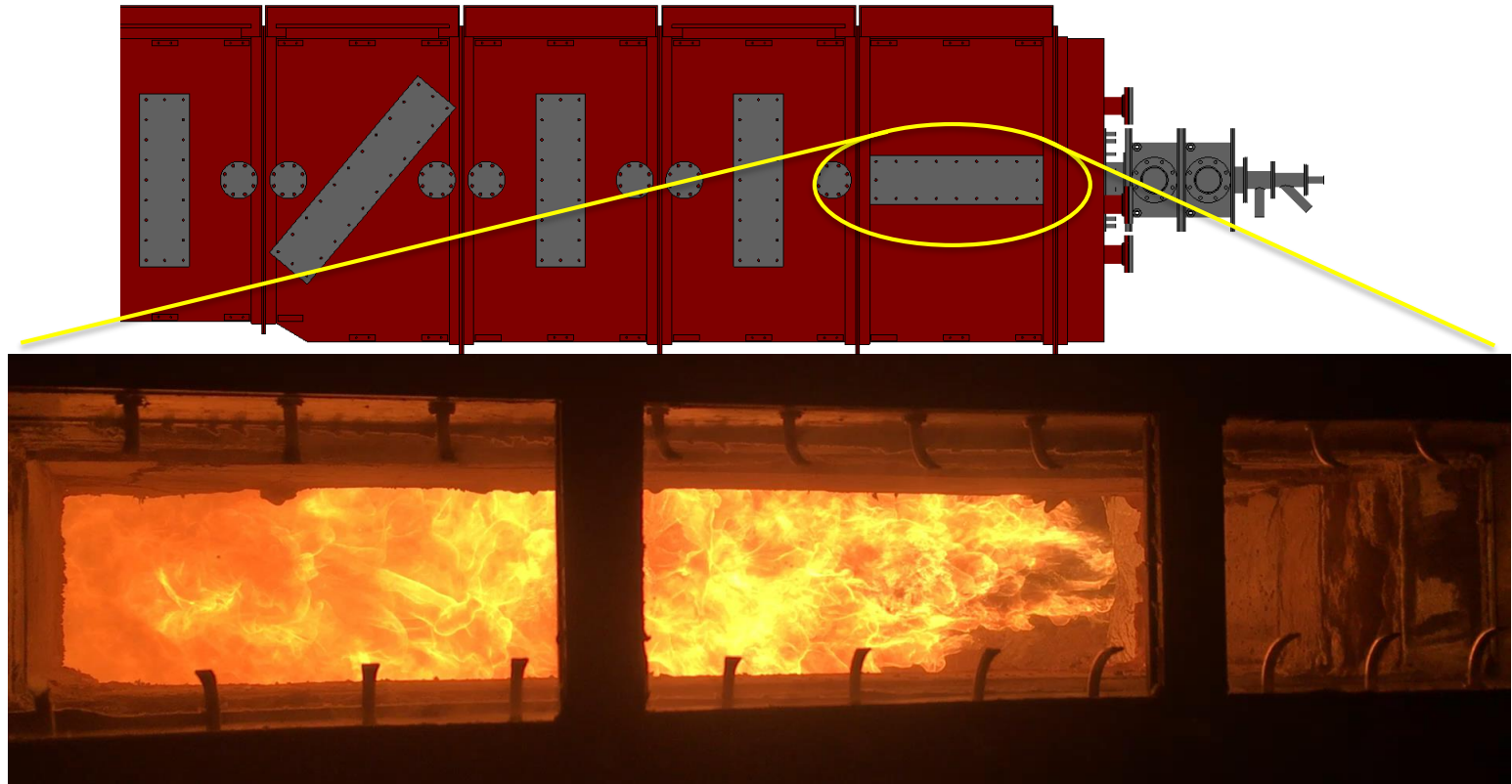


■ - Primary Gas / Coal

■ - Secondary Gas (O<sub>2</sub>)

# L1500 Heat Balance

Example Data Set



\* Air-fired flame at the end of the high temperature oxygen test

# L1500 Heat Balance

## Example Data Set

### Methods:

- Furnace heat removal can be assessed in two ways
  - Enthalpy of the reactants minus the enthalpy of the flue gas at the furnace exit
  - Direct measurement of active heat removal through water cooled surfaces plus heat loss through the refractory wall



# L1500 Heat Balance

## Example Data Set

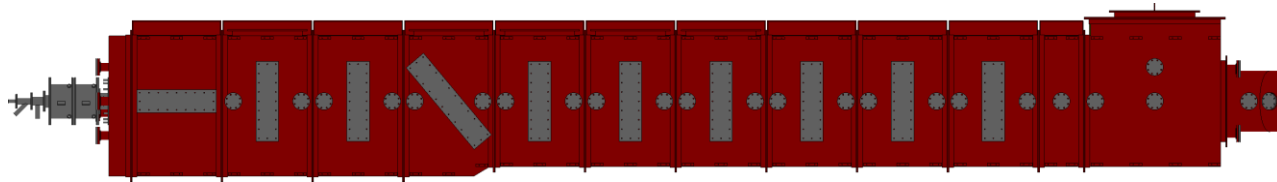
### Assumptions:

- Heat loss through the refractory wall is significant
  - Can be estimated using the measured heat flux in the roof of each section.
  - Heat loss is assessed by applying the measured heat flux uniformly across each furnace section
  - Heat flux through the burner plate is assumed to be the same as in section 1
  - Heat flux through section 11 and 12 is assumed to be the same as section 10
  - Heat removal through both radiation heat exchangers is assumed to be the same.

# L1500 Heat Balance

Example Data Set

Preheated Gas  
0.01 MMBtu/hr



Flue Gas  
0.33 MMBtu/hr

Cooling Panels

0.59  
MMBtu/hr

Cooling Coils

0.94  
MMBtu/hr

Cooling Jackets

0.31  
MMBtu/hr

Wall Heat Loss

0.80  
MMBtu/hr

Heat Loss From Furnace  
2.69 MMBtu/hr

←→  
1.3 % Difference

Measured Heat Removal  
2.64 MMBtu/hr

## Summary & Conclusions

- Weaknesses of year 1 measurements performed in the 1.5 MW oxy-coal unit have been identified
- Measurement devices have been upgraded to quantify:
  - Heat transfer through cooling surfaces
  - Wall temperatures
  - Radiation intensity
- Instrument models have been developed
- Pathway for uncertainty quantification has been developed



# Questions